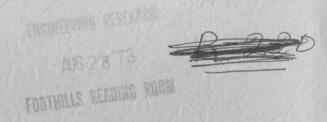
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CALIBRATION OF A VENTURI TUBE

FOR

MODEL PUMP TEST

Prepared for Fairbanks-Morse, Inc.

Colorado State University
Engineering Research Center
Fort Collins, Colorado

January 1965

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CALIBRATION OF VENTURI TUBE FOR MODEL PUMP TEST

General

Calibration of a 16 x $10\frac{1}{2}$ in. Simplex Type VT No. T8757 venturi tube was authorized through issuance of Purchase Order No. 155085-3 dated October 21, 1964 by Colt Industries, Inc., Fairbanks-Morse, Inc. Pump Division, Kansas City, Kansas.

The pump model test at Kansas City required that--

"--The model discharge shall be measured directly by accurately calibrated gravimetric or volumetric equipment; or indirectly by means of a venturi tube, --provided the device used is calibrated in situ by gravimetric or volumetric equipment. The equipment or device as installed shall be capable of indicating and responding to a change in discharge not greater than plus or minus 0.5 percent of the model discharge at the range of discharge between 50 and 100 percent. Also, the maximum variance of the device at any model discharge, within the above range, shall not exceed plus or minus 0.5 percent from the true discharge, as determined by plotting the indicated values against true values. Proper corrections shall be made where necessary for the effect of temperature, barometric pressure, gravity, and local water density--."

In view of the requirement quoted above, one section of the test loop at the Kansas City Laboratory of Faribanks-Morse was shipped and adapted to the calibration facility at the Engineering Research Center at Colorado State University. A schematic drawing of the calibration facility at CSU, including the portion of the test loop is shown in Fig. 1.

The Venturi Tube

The venturi tube is a well-known flow meter. The discussion herein is therefore intended only to lend completeness to this report. The venturi tube is a device to accelerate the fluid and temporarily lower its static pressure. Suitable pressure connections are provided to measure the magnitude of the reduced static pressure at the constricted section. A typical form of the hydraulic gradient is shown in Fig. 2. In the figure, h is the difference in the levels of the free fluid column attached to points 1 and 2, which is the difference in static pressures measured at the conduit wall. The magnitude of h is subject to variation depending upon the velocity of flow, the acceleration through the tube and the friction and form losses between sections 1 and 2.

Writing the Bernoulli equation along a streamline, in this case, the centerline of the pipe and venturi tube,

$$\frac{V_1^2}{2g} + \frac{p_1}{og} = \frac{V_2^2}{2g} + \frac{p_2}{og} + h_f \tag{1}$$

where V = velocity at the point under consideration in ft/sec,

p = pressure at the same point in lb/ft²,

g = local gravitational acceleration, ft/sec²,

 ρ = fluid mass density, lb-sec²/ft⁴,

h
f = energy loss per pound of fluid between point 1 and 2,
ft-lb/lb,

subscripts 1 and 2 refer to successive points along the stream line. From Fig. 2, and Eq. (1)

$$h = \frac{1}{\rho g} (p_1 - p_2) = \frac{1}{2g} (V_2^2 - V_1^2) + h_f$$
 (2)

For physical convenience, pressures are measured at the walls of the conduit rather than at the centerline. In practice therefore, differences in wall pressures are measured, rather than pressures at the centerline, and a calibration is generally performed to determine the relationship of h to volume flow rate Q. Where accurate flow measurement is desired, the venturi tube as all other flow meters, must be calibrated periodically when changes in wall roughness, or otherwise small changes in flow geometry are suspect.

The calibrated relationship of h to Q can most conveniently be expressed in graphical form. However, where a meter is calibrated at one location and is used at another, the relationship of h to Q determined at the calibration site requires correction at the location of use because

of change in local gravitational acceleration and specific weights of the fluid. To be accurate within the limits specified heretofore, it is more convenient to determine a discharge coefficient for the flow meter. The discharge coefficient may be expressed as

$$C = \frac{\text{actual mass flow rate } \rho Q_{a}}{\text{theoretical mass flow rate } \rho Q_{t}}$$
(3)

or where the mass density is a constant,

$$C = \frac{Q_a}{Q_+} \tag{4}$$

where Q_a = actual volume rate of flow, Q_+ = theoretical volume rate of flow.

The coefficient of discharge is a function of wall roughness, conduit geometry, and flow and fluid properties. It can thus be expressed more conveniently as a function of Reynolds number, $R = \frac{V_2 d_2}{\nu}$, where d_2 is the venturi tube throat diameter.

Expressing the theoretical discharge in terms of the throat diameter and velocity,

$$Q_{t} = A_{2} \sqrt{2g \left(\frac{P_{1} - P_{2}}{\rho g}\right)} \left(\frac{1}{\sqrt{1 - \left(\frac{A_{2}}{A_{1}}\right)}}\right)$$
 (5)

Let

$$\frac{A_2}{A_1} = \left(\frac{d_2}{d_1}\right)^2 = \beta^2 ,$$

So that
$$Q_t = A_2 \sqrt{\frac{2\Delta p}{\rho}} \qquad \left(\frac{1}{-\sqrt{1-\beta^4}}\right)$$
 (6)

For the venturi meter calibrated the following quantities were used:

$$d_{1} = 16 \text{ in.}$$

$$d_{2} = 10.5 \text{ in.}$$

$$\pi d_{2}^{2} = 86.588 \text{ in.}^{2} = 0.6013 \text{ ft}^{2}$$

$$\beta^{4} = \left(\frac{10.5}{16}\right)^{4} = .18547$$

$$g = 32.1441 \text{ ft/sec}^{2}$$

- p of water varies with temperature, see Fig. 3.
- $\gamma = \rho g$, of water varies with temperature, see Fig. 3.
- $\nu = \frac{\mu}{2}$ of water varies with temperature, see Fig. 4.
- S = specific gravity of barium = 2.95.

Calibration Procedure

Water flow through the calibration system was supplied by a Fairbanks-Morse turbine pump. Discharge through the venturi was controlled by the valve downstream of the venturi tube shown in Fig. 1. At each change of discharge, sufficient time was allowed to establish steady flow. Pressure differential was measured with a differential barium-water manometer and a water-air differential manometer, which was also sloped for greater accuracy at the low discharges.

Discharge measurements were made volumetrically in a calibrated tank and with a calibrated clock measuring time to the nearest 0.01 second. The volumetric tank was calibrated by the weight method, and the scale used for this purpose was in turn calibrated with standard weights checked by the U.S. Bureau of Standards at Boulder, Colorado. The specific weight of vater used in the calibration of the tank was corrected for temperature and local gravitational acceleration. Weight of water was corrected for air buoyancy. The clock or timer used was calibrated against standard time signal broadcast by the N.B.S., and the electrical line frequency at the laboratory was checked against N.B.S. traceable standard frequency and was found to be 60 00 c.p.s. The accuracy of the calibration system is given in the following table:

Table of Calibration Fixture Accuracies

Item	Method of Reading	Reading Accuracy	Accuracy Absolute Units	Total Capacity	Accaracy Percent
Large Volumes	Hook Gage	+0.001 ft	+2.0 gal	8800 gal	.022
Small Volumes	Hook Gage	+0.001 ft	+1.0 gal	3850 gal	•□25
Time	Electric Clock		<u>+</u> •Ol sec	60 sec. (minimum)	•316
			Total	Accuracy	.)41 percent

There is another possible source of error in the diverting mechanism of the calibration stand. The diverting equipment is designed to operate with balanced time intervals between diverting flow into the tank and into the wasteway (see Fig. 1). The total time interval for a swing one way requires 1.4 seconds. Assuming that an error of .1 sec could occur in the swing time intervals and with maximum discharge through the calibration system, of 10,000 gpm, an error of 16.7 gallons can occur in the volumetric increment relative to the correct time interval. Thus an additional percent error of 0.189 could occur. Adding this to the accuracy of the fixture gives an overall accuracy of 0.227 percent, well within the ±0.5 percent required for the pump model tests.

Calibration Results

The results of the calibration are shown in Fig. 5 with discharge Q as a function of pressure differential Δp in psi, and discharge coefficient C as a function of Reynolds No. in Fig. 6. The calibration data are given in a separate table appended to this report. The curve in Fig. 5 is of course too small in scale to read within the desired accuracy of +0.5 percent.

In Fig. 6, an average curve in solid line is drawn "by eye" through the data points. A dashed line is also drawn parallel to the curve to indicate the limits of the ±0.5 percent accuracy. It will be noted that all data points can be made to lie within the band, except at the lowest discharge, where the accuracy appears to be within ±0.7%. Also drawn as a broken line curve on Fig. 6 is an approximate "theoretical" curve offered by The Simplex Valve and Meter Co. for this valve on May 14, 1943. The term approximate is used here because the throat Reynolds numbers cannot be calculated without assuming a water temperature. It is to be noted that the "theoretical" curve lies within the 0.5 percent band.

As a supplement to the calibration of this venturi tube, several velocity profiles were taken in the approach pipe to the venturi tube at different discharges. The purpose of these profiles was to show that the flow conditions under which the calibration were made represented the flow conditions at the laboratory where the pump tests were to be performed. The velocity profiles taken at the Engineering Research Center of Colorado

State University are shown in Figures 7, 8 and 9. For comparison purposes, the velocity profiles taken at the Kansas City Laboratory are shown in Figs. 10, 11, and 12. Although the discharges do not correspond exactly, suitable comparisons can be made.

A Cole pitot meter supplied by the Fairbanks-Morse Company was used to make the velocity measurements. A differential Pace pressure transducer was used with the pitometer. The pressure transducer was calibrated at the site at the water temperature during the velocity measurements. The rating curve for the Cole pitometer was supplied by the Fairbanks-Morse Company and is shown as Fig. 13. The data and computations for the velocity traverses are appended.

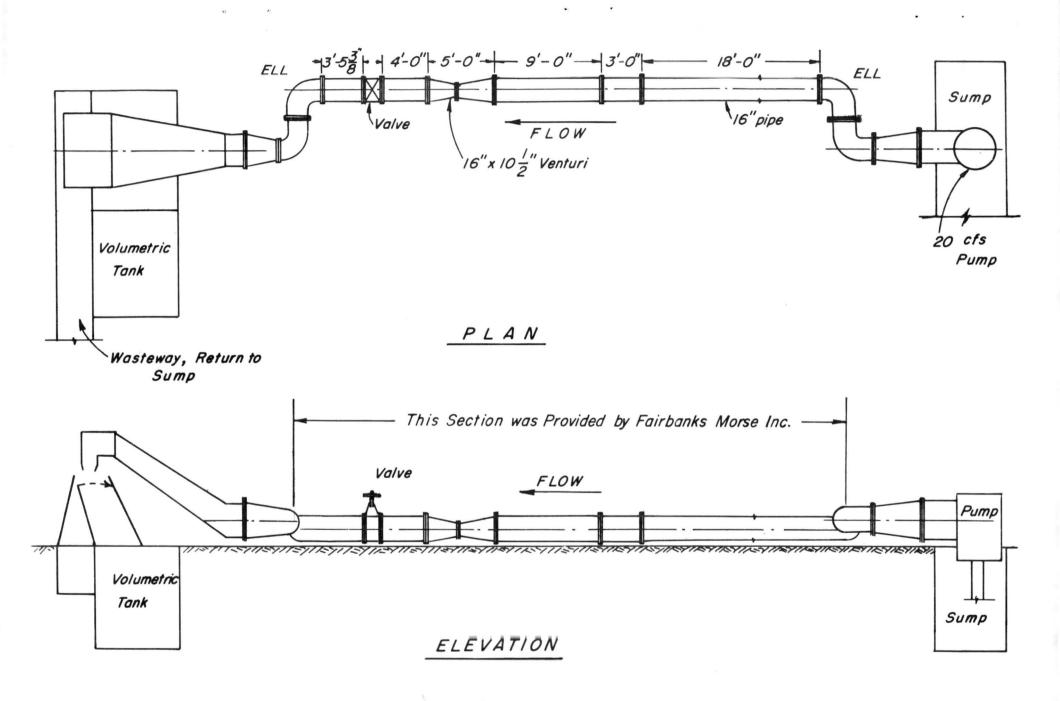


FIGURE I SCHEMATIC DRAWING OF CALIBRATION FACILITY

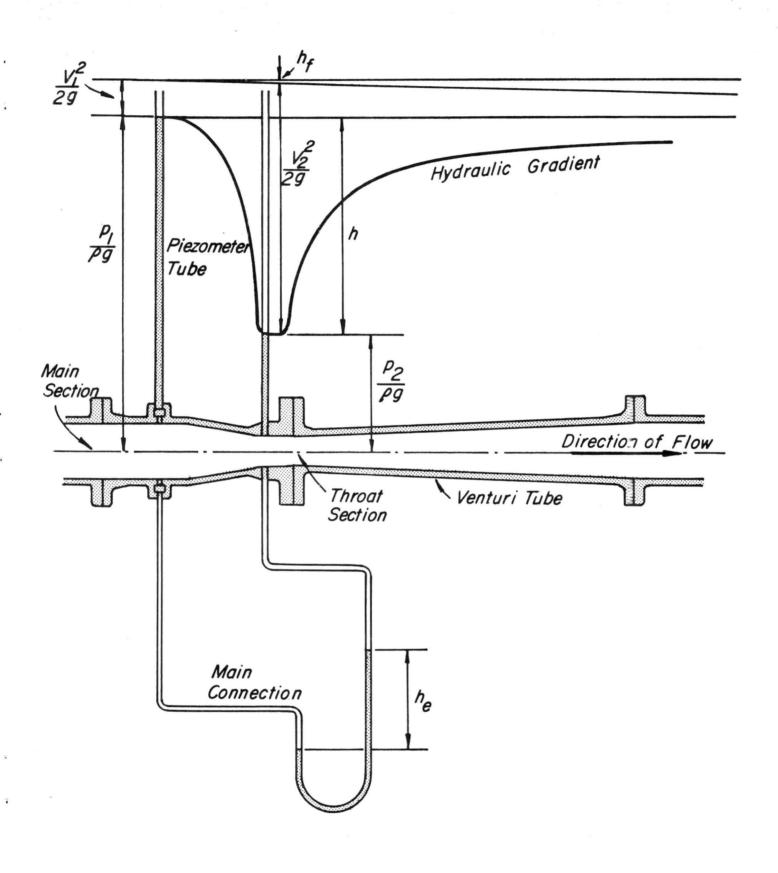


FIGURE 2 TYPICAL HYDRAULIC GRADIENT THROUGH YENTURI TUBE

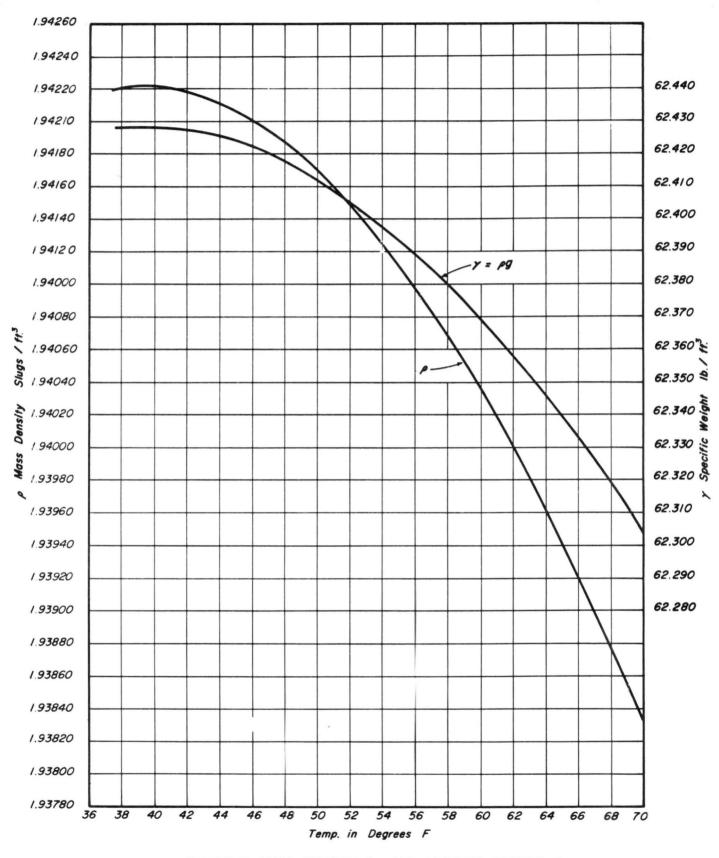


FIGURE 3 MASS DENSITY & AND SPECIFIC WEIGHT Y

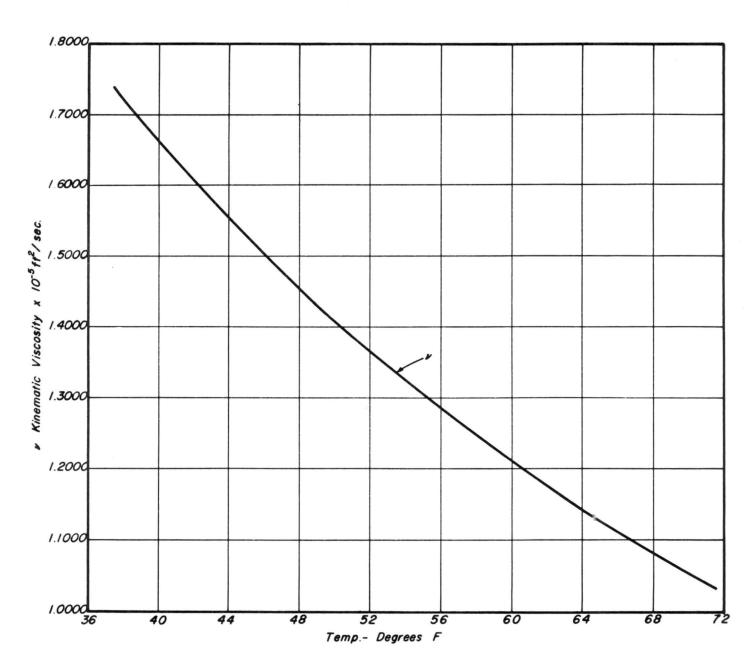


FIGURE 4 KINEMATIC VISCOSITY OF WATER

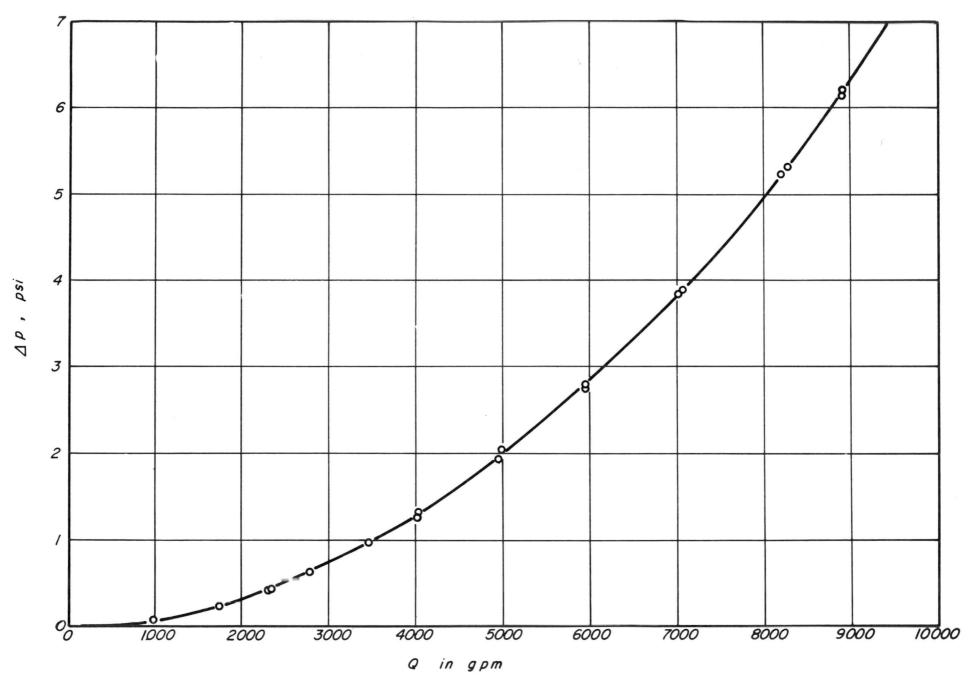


FIGURE 5 VARIATION OF DISCHAGE WITH DIFFERENTIAL PRESSURE

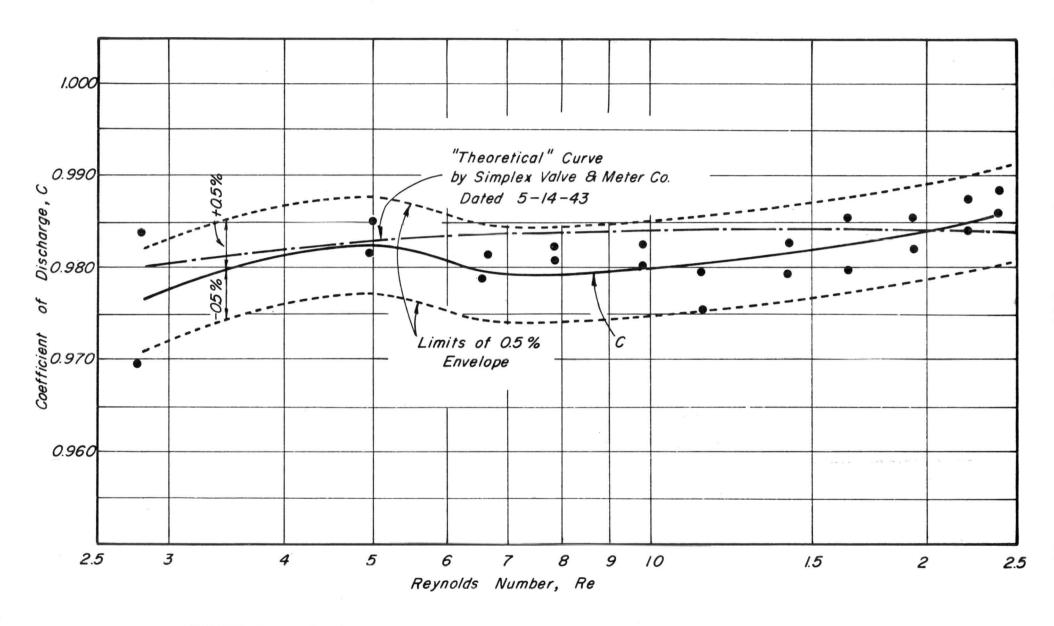


FIGURE 6 VARIATION OF DISCHARGE COEFFICIENT C WITH REYNOLDS NUMBER

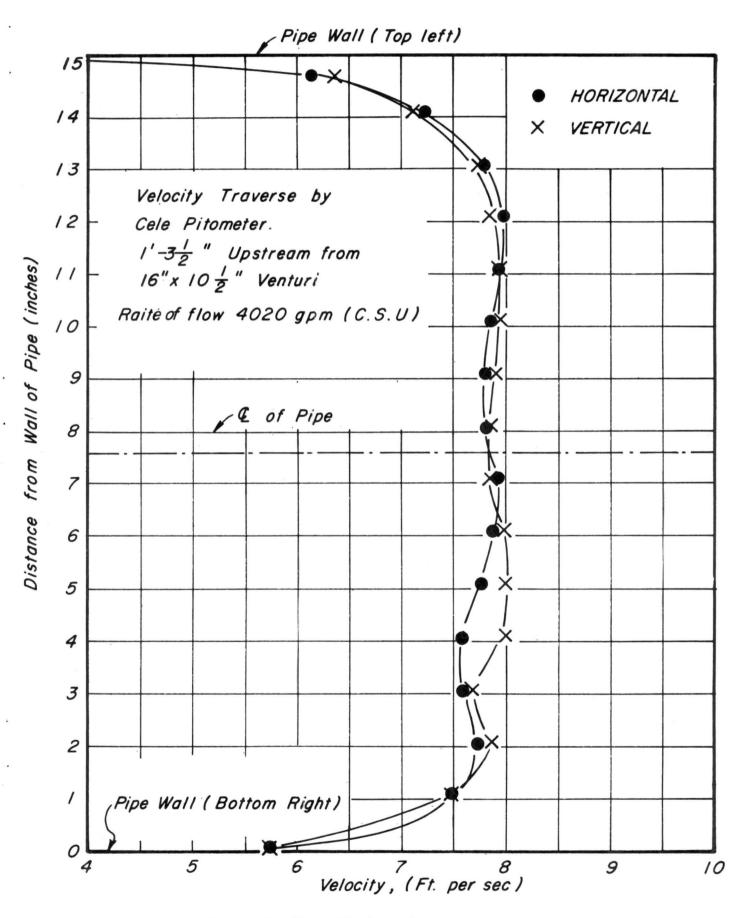


FIGURE 7 VELOCITY PROFILES

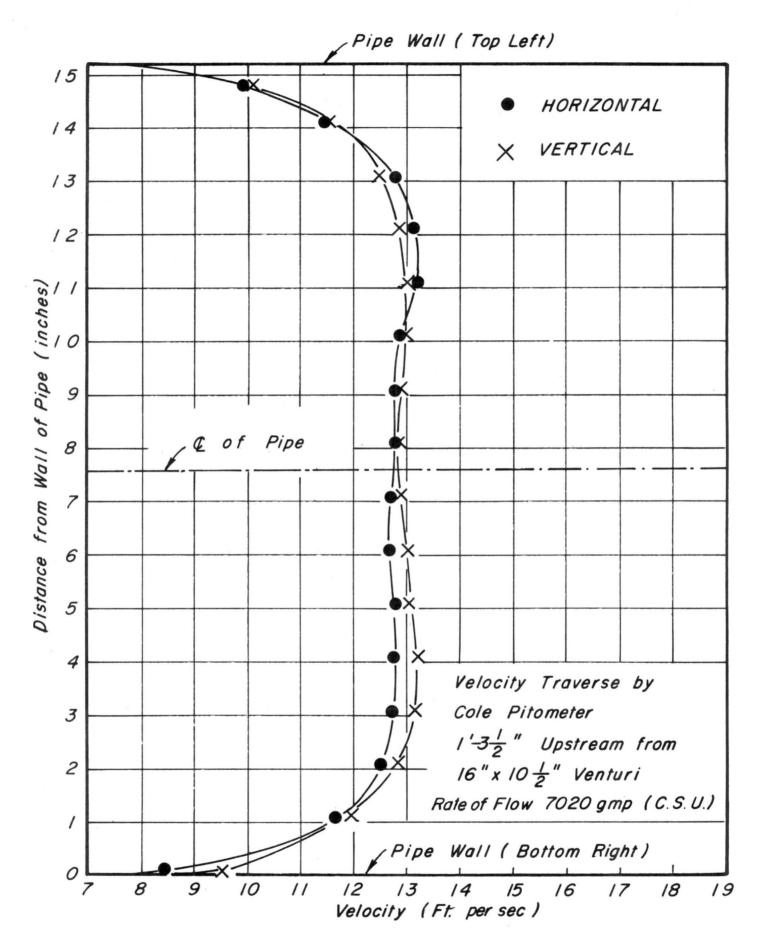


FIGURE 8 VELOCITY PROFILE

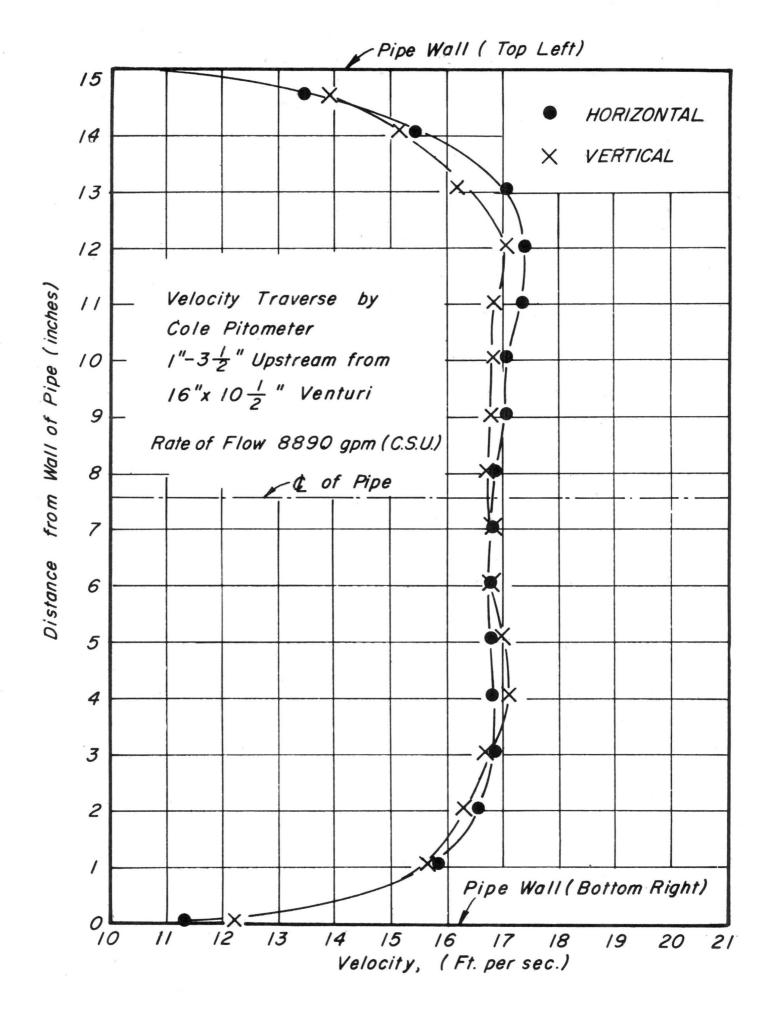
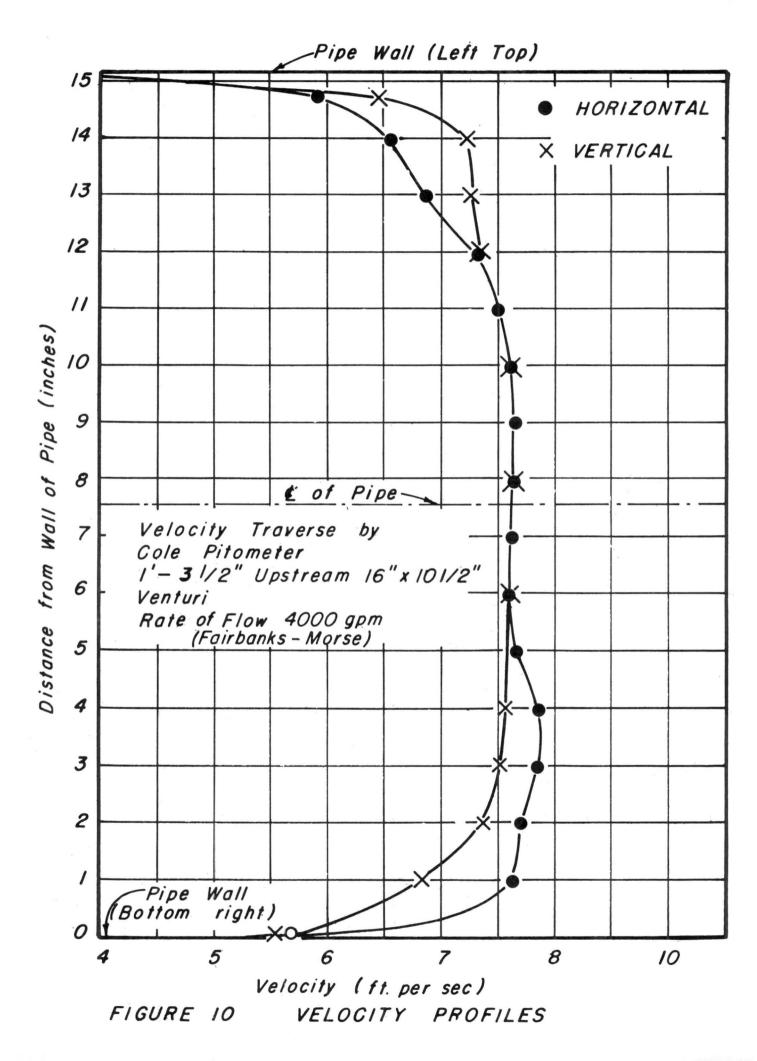
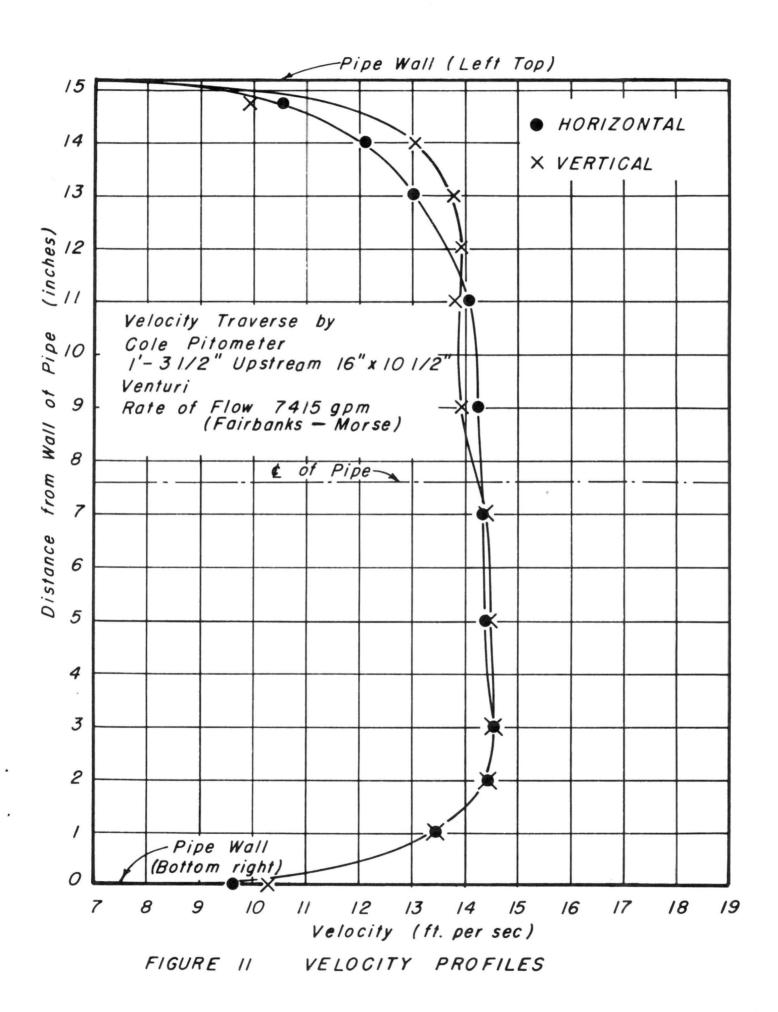


FIGURE 9 VELOCITY PROFILES





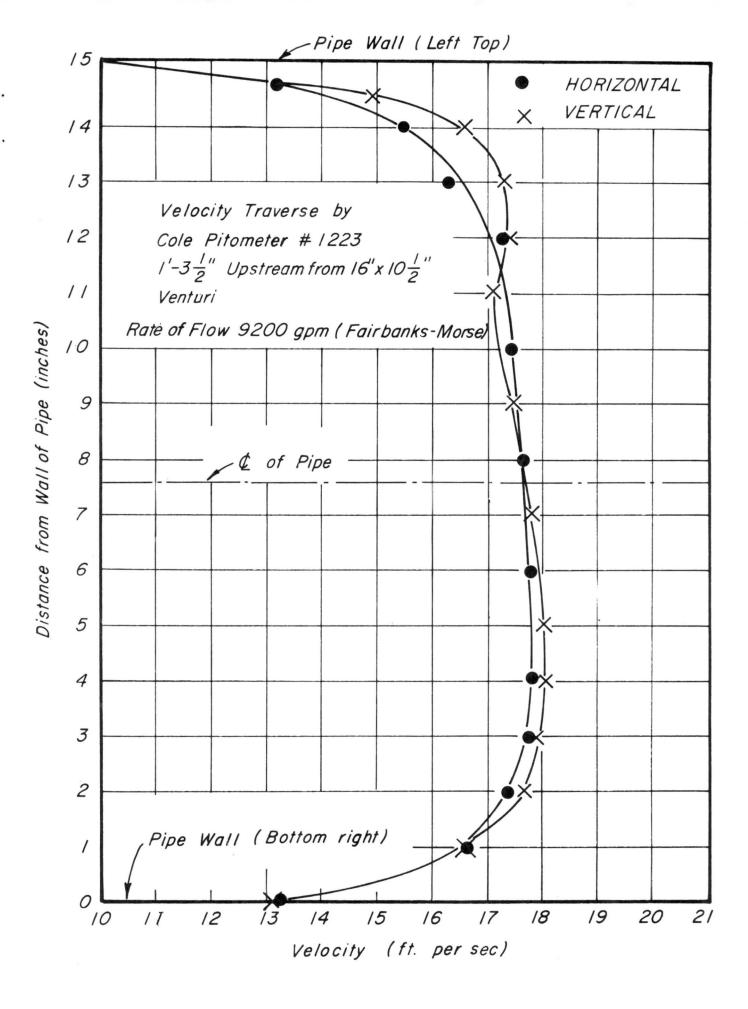


FIGURE 12 VELOCITY PROFILES

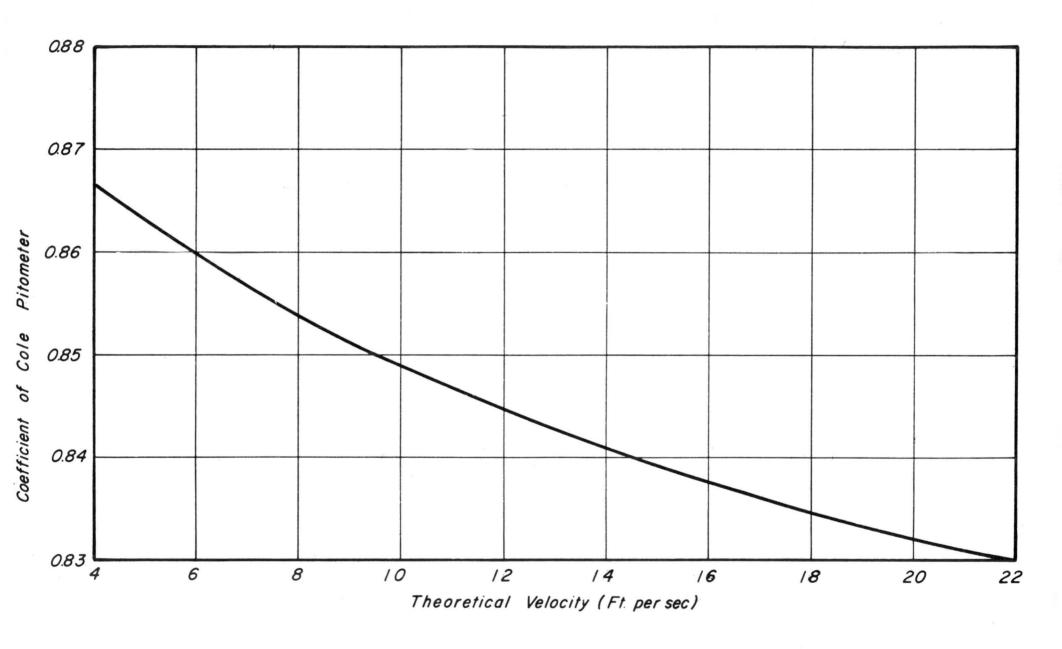


FIGURE 13 COLE PITOMETER RATING CURVE

APPENDIX

Tables of Data and Computation

TABLE 1. CALIBRATION DATA AND CALCULATIONS FOR FIGS. 5 AND 6.

Run No.	Time (Min)	Water Temp. (°F)		Δh nometer Ft of Wate	Vol. Gal	γ	Δp psi.	Δp psi.	ρ	<u>Δp</u> ρ	$\sqrt{\frac{\Delta p}{\rho}}$	Meas. Flow Rate gpm	Theo. Flow Rate gpm	С	v ₂	v ₂ d ₂	ν	Re
1	0.9997	60.0	7.278	14.192	8,927.29	62.369	6.147	885.14	1.94035	456.18	21.358	8,929.97	9,032.73	.9886	33.086	28.950	1.2111.105	2.3904.106
2	1.0002	60.0	7.293	14.221	8,918.67	62.369	6.159	886.95	1.94035	457.11	21.380	8,916.89	9,042.03	.9862	33.038	28.908		2.3869
3	1.1337	60.0	6.284	12.254	9,397.54	62.369	5.307	764.27	1.94035		19.846	8,289.27	8,393.27	.9876	30.671	26.371	1.2111	2.2159
14	1.0835	60.5	6.212	12.113	8,898.66	62.366	5.246	755.44	1.94027	389.35	19.732	8,212.88	8,345.06	.9842	30.388	26.590		2.2145
5	1.1000	61.0	4.586	8.943	7,772.47	62.363	3.873	557.71	1.94018	287.45	16,954	7,065.88	7,170.19	.9855	26.144	22.876		1.9162
6	1.2503	62.0	4.555	8.882	8,775.13	62.358	3.846	553.86	1.93999	285.50	16.897	7,018.42	7,146.08	.9821	25.969	22.723		1.9306
7	1.5003	62.0	3.296	6.427	8,935.76	62.358	2.783	400.77	1.93999	206.58	14.373	5,955.98	6,078.63	.9798	22.038	19.283		1.6383
8	1.4983	62.0	3.259	6.355	8,926.67	62.358	2.752	396.29	1.93999	204.27	14.292	5,957.87	6,044.37	.9857	22.045	19.289		1.6388
9	1.9170	64.0	2.285	4.458	9,538.26	62.345	1.930	277.93	1.93959	143.29	11.970	4,975.62	5,062.35	.9829	18.410	16.109		1.4075
10	2.0000	64.0	2.287	4.460	9,918.99	62.345	1.931	278.06	1.95959		11.973	4,959.50	5,063.62	.9794	18.351	16.057		1.4030
11	2.5007	64.0		2.969	10,079.96	62.345	1.285	185.10	1.95959	95.432		4,030.86	4,131.46	-9756	14.915	13.051		1.1403
12	2.5007	64.0		2.936	10,065.69	62.345	1.271	183.04	1.95959	94.370	9.7144	4,025.15	4,108.41	•9797	14.893	13.031		1.1386 .7897
13	3,3518	64.0		1.401	9,363.59	62.345	0.607		1.95959	45.032	6.7196	2,793.60	2,841.85	.9830	10.337	-	1.1445	.7856
14	3.5512	64.0		1.392	9,868.81	62.345	0.603		1.95959	44.743	6,6890	2,779.01	2,828.91	.9824	10.283		1.1445	.6502
15	4.1523	64.0		0.968	9,551.32	62.345	0.419		1.95959	31.115	5.5781	2,300.25	2,359.09	•9751	8.511		1.1445	.6528
16	1.6673	64.0		0.968	3,850.48	62.345	0.419		1.95959	31.115		2,309.41	2,359.09	.9789	8.545		1.1445	.7842
17	3.4840	64.0		1.392	9,665.44	62.345	0.603		1.95959	44.743	6.6890	2,774.24	2,828.91	.9807	10.265		1.1445	.9776
18	2.7487	64.0		2.165	9,507.00	62.345	0.937	134.98	1.95959	69.592	8.3422	3,458.73	3,528.08	.9803	12.798	11.198		
19	2.8340	64.0		2.157	9,804.96	62.345	0.934	134.48	1.95959	69.334	8.3267	3,459.76	3,521.53	.9825	12.801	11.201		•9779 •2799
20	4.0000	64.0		0.1811	3,960.68	62.345	0.0784		1.95959	58.213	2.4129	990.17	1,020.46	•9703	3.664		1.1445	·2790
21	4.0008	64.0		0.1772	3,948.97	62.345	0.0769		1.95959	56.960	2.3720	987,05	1,003.17	.9839	3.652		1.1445	.4948
22	2.2503	64.0		0.5529	3,939.14	62.345	0.239		1.95959	17.772	4.2157	1,750.50	1,782.90	.9818	6.477		1.1445	.4974
23	2.2505	64.0		0.5530	3,952.74	62.345	0.239		1.95959	17.775	4.2161	1,756.38	1,783.07	.9850	6.499		1.1445	.6628
24	3.9853	64.0 64.0		0.9938	9,348.42	62.345	0.430		1.95959	31.944	5.6519	2,345.73	2,390.30	.9814	8.677 3.614		1.1445	.2761
25	4.0002	64.0		0.1765	3,907.06	62.345	0.0764		1.95959	56.733	2.3819	976.72	1,007.35	.9696 .9855	4.851	12.995		1.1345
26	2.5052	04.0		2.885	9,573.14	62.345	1.249	179.87	1.95959	92.736	9.6300	4,013.56	4,072.72	.9077	4.051	T5.332	1.1447	

TABLE 2. COLE PITOT TUBE SURVEY

Q = 4020 gpm, Water Temperature 64°F

Pt.	Dist. In.	∆h	- √∆h	Theo. Vel.	С	Act. Vel.	Remarks
Right	to Left	;					
1 2 3 4 5 6 7 8 9 0 11 12 14 15 16	0.09 1.09 2.09 3.09 4.09 5.09 6.09 7.09 8.09 9.09 11.09 12.09 13.09 14.77	0.55 1.20 1.28 1.27 1.27 1.29 1.33 1.35 1.30 1.30 1.30 1.36 1.36 1.39 1.30	.74 1.09 1.13 1.125 1.125 1.135 1.152 1.16 1.14 1.15 1.165 1.169 1.14 1.052 0.895	5.93 8.75 9.07 9.03 9.11 9.31 9.15 9.15 9.35 9.15 9.35 9.15 9.35 9.15 9.35 9.15 9.35 9.15 9.35 9.15 9.35 9.15 9.35 9.15 9.15 9.15 9.15 9.15 9.15 9.15 9.1	.860 .852 .851 .851 .851 .850 .851 .850 .851 .850 .851	5.10 7.45 7.71 7.57 7.57 7.75 7.86 7.78 7.85 7.94 7.97 7.78 7.94 7.97	Right side Left side
Botto	m to Top)					
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16	0.09 1.09 2.09 3.09 4.09 5.09 6.09 7.09 8.09 9.09 11.09 12.09 14.09 14.64	0.688 1.18 1.32 1.26 1.38 1.37 1.37 1.32 1.32 1.34 1.36 1.35 1.36 1.35	.83 1.085 1.15 1.12 1.17 1.17 1.17 1.15 1.156 1.165 1.16 1.15 1.16 1.15	6.65 8.71 9.23 8.99 9.39 9.39 9.23 9.27 9.27 9.31 9.07 8.34	.858 .852 .851 .850 .850 .850 .851 .851 .850 .851 .855	5.71 7.42 7.85 7.65 7.99 7.99 7.85 7.89 7.95 7.92 7.12 6.36	Top

TABLE 3. COLE PITOT TUBE SURVEY Q = 7020 gpm, Water Temperature $64^{\circ}F$

Pt.	Dist. In.	Δh	- √∆h	Theo. Vel.	C	Act. Vel.	Remarks
Right	to Left						
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16	0.09 1.09 2.09 3.09 4.09 5.09 6.09 7.09 8.09 9.09 10.09 11.09 12.09 14.09	1.54 2.98 3.42 3.58 3.60 3.58 3.58 3.61 3.66 3.80 3.80 3.80 3.80 3.80 3.80 3.80 3.80	1.24 1.725 1.85 1.897 1.899 1.88 1.89 1.899 1.92 1.96 1.95 1.69 1.464	9.95 13.84 14.85 15.17 15.21 15.23 15.09 15.17 15.23 15.41 15.73 15.65 15.65 15.58 11.75	.8395 .8386 .8388 .839 .8386 .8388 .8388 .8385 .638 .8381 .8389	8.45 11.64 12.48 12.71 12.78 12.80 12.67 12.71 12.80 12.80 12.94 13.20 13.12 12.77 11.42 9.93	Right Left side
Botto	om to Top	2					
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16	0.09 1.09 2.09 3.09 4.09 5.09 6.09 7.09 8.09 9.09 10.09 11.09 12.09 14.09 14.79	1.97 3.151 3.80 3.85 4.74 4.66 6.66 4.77 4.94 4.94	1.402 1.775 1.90 1.95 1.96 1.93 1.91 1.91 1.93 1.94 1.91 1.86 1.71 1.495	11.25 14.23 15.25 15.64 15.72 15.5 15.33 15.33 15.33 15.33 15.5 15.57 15.57 15.33 14.92 13.71	.8405 .8388 .8381 .8385 .8385 .8386 .8386 .8386 .8386 .8386 .8393 .8414	9.51 11.96 12.80 13.13 13.20 13.01 13.01 12.88 12.88 13.01 13.07 12.88 12.52 11.53 10.14	Тор

TABLE 4. COLE PITOT TUBE SURVEY Q = 8890 gpm, Water Temperature $64^{\circ}F$

Pt.	Dist. In.	Δh	$-\sqrt{\Delta h}$	Theo. Vel.	С	Act. Vel.	Remarks
Right	to Left						
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16	0.09 1.09 2.09 3.09 4.09 5.09 6.09 7.09 8.09 9.09 11.09 12.09 14.09 14.79	2.75 5.62 6.10 6.33 6.26 6.26 6.26 6.45 6.45 6.45 6.45 5.27	1.67 2.37 2.47 2.52 2.50 2.50 2.50 2.50 2.54 2.60 2.60 2.54 2.60 2.99	13.4 19.0 19.8 20.2 20.1 20.1 20.1 20.4 20.4 20.4 20.8 20.8 20.4 18.4 16.0	.842 .833 .832 .832 .832 .832 .832 .832 .83	11.3 15.8 16.5 16.75 16.75 16.75 16.75 16.75 17.0 17.3 17.3 17.3 17.3 17.3	Right
Botto	m to Top)					
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